

# Thermoeconomic analysis of recuperative sub- and transcritical organic Rankine cycle systems

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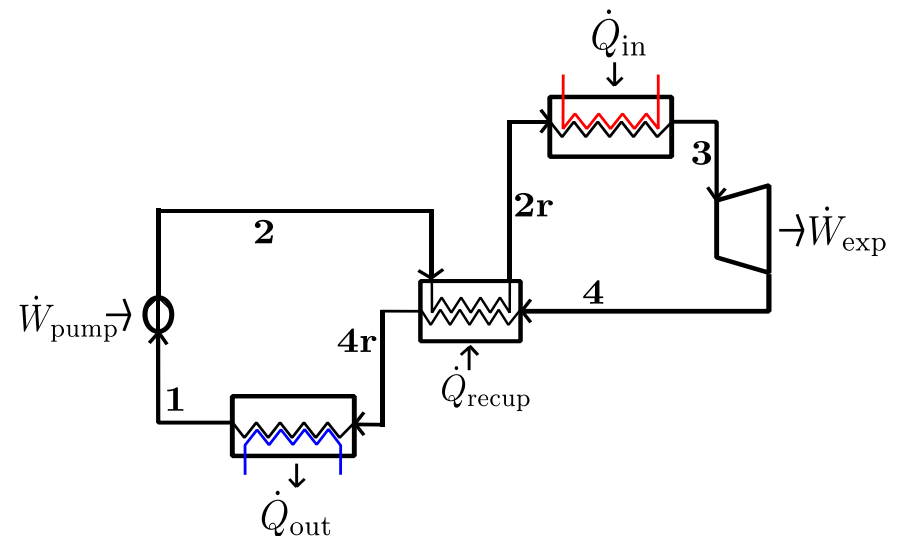
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## Outline

- Introductory background – ORC system configurations/architectures
  - Model and optimization algorithm
  - Results – Effect of evaporation pressure
  - Results – Subcritical and transcritical systems
  - Results – Condenser boundary conditions
  - Conclusions
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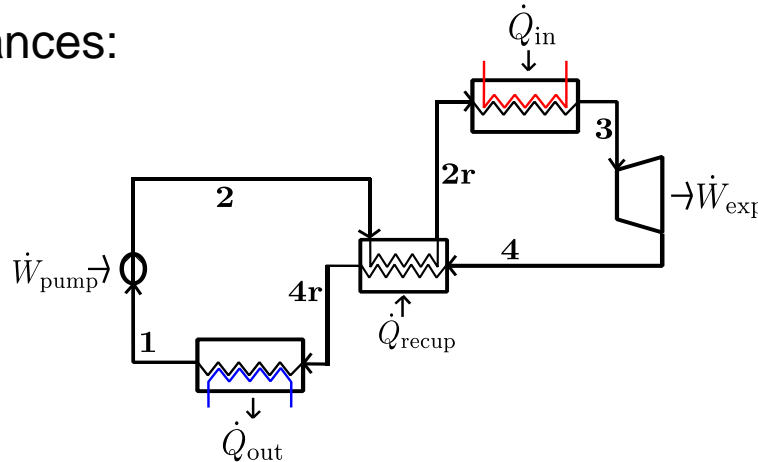
# ORC system configurations/architectures

- Subcritical systems
- Transcritical systems
- Trilateral, partial evaporative, and other systems
- Recuperative ORC systems?



# Model and algorithm – thermodynamic model

- Energy balances:



- Assumptions:  
 $\eta_{s,exp} = 75\%$   
 $\eta_{s,pump} = 85\%$   
 $\Delta T_{min} = 10\text{ }^{\circ}\text{C}$

- Recuperator:

$$\theta_{recup} = \frac{T_4 - T_{4r}}{T_4 - (T_2 + \Delta T_{min})} \approx \frac{h_4 - h_{4r}}{h_4 - h(T_2 + \Delta T_{min}, P_{evap})}$$

$\theta_{recup} \cong 0 \rightarrow \rightarrow$  No recuperation,  $T_4 = T_{4r}$ ,  $T_2 = T_{2r}$

$\theta_{recup} > 0 \rightarrow \rightarrow$  Recuperator deployed.

## Model and algorithm – costing model

- Pump, pump motor and expanders are costed using the correlations proposed by Seider *et al.* and Turton *et al.*:

$$C_p^0 = F \exp(Z_1 + Z_2 \ln X + Z_3 \ln(X)^2 + Z_4 \ln(X)^3 + Z_5 \ln(X)^4)$$

$$C_p^0 = F 10^{(Z_1 + Z_2 \log X + Z_3 \log(X)^2)}$$

$X$  the sizing attribute (power, heat-transfer area etc.)

$F, Z_n$  correlation coefficients

- Heat exchangers costed with the ‘C-value’ method and the ESDU 92013 chart
- Equipment cost modified to account for high/low pressure effects
- Costs converted to today’s prices using the CEPCI

# Boundary conditions and problem definition

- Heat source: Waste-heat flue gas in @ 380 °C and 185 kg/s
- Heat sink: Cooling water in @ 25 °C; out @ 55 °C for a  $\Delta T$  of 30 °C
- Working fluids (34 in total):
  - 8 × Light hydrocarbons (butane, isobutane, pentane, ...)
  - 5 × Heavy & aromatic hydrocarbons (benzene, nonane, decane, ...)
  - 16 × Refrigerants (R218, R227ea, R245fa, ...)
  - 5 × Siloxanes (D4, D5, MM, MDM, MD2M)

\*Thermodynamic properties from Refprop

## Boundary conditions and problem definition

- Optimization problem:

$$\text{maximize } \dot{W}_{\text{net}}$$

$$\text{subject to: } \Delta T_i \geq \Delta T_{\text{min}}(10 \text{ }^\circ\text{C}) \quad \forall i$$

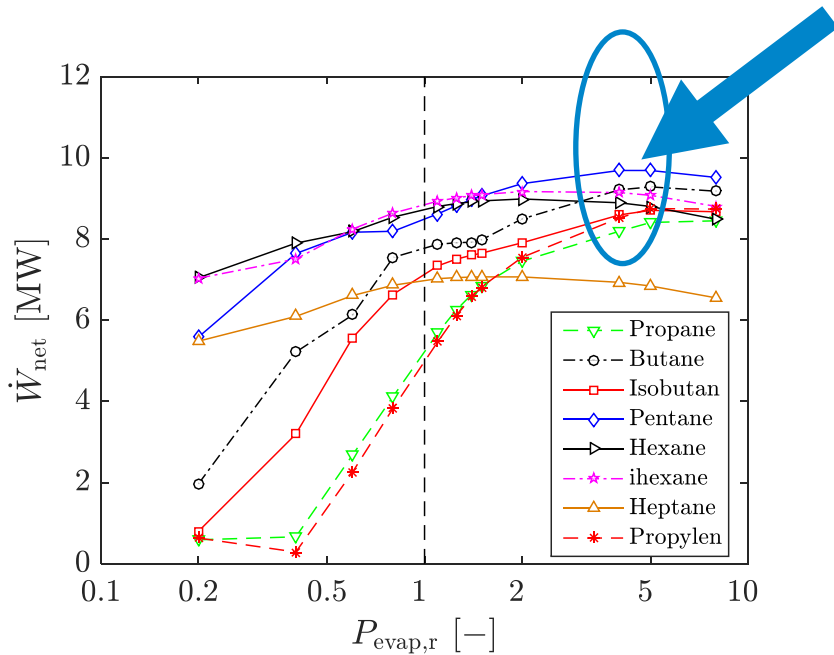
$$T_4 \geq T_{\text{dew}}(P_{\text{cond}})$$

$$0 \leq \theta_{\text{SH}}, \theta_{\text{recup}} \leq 1$$

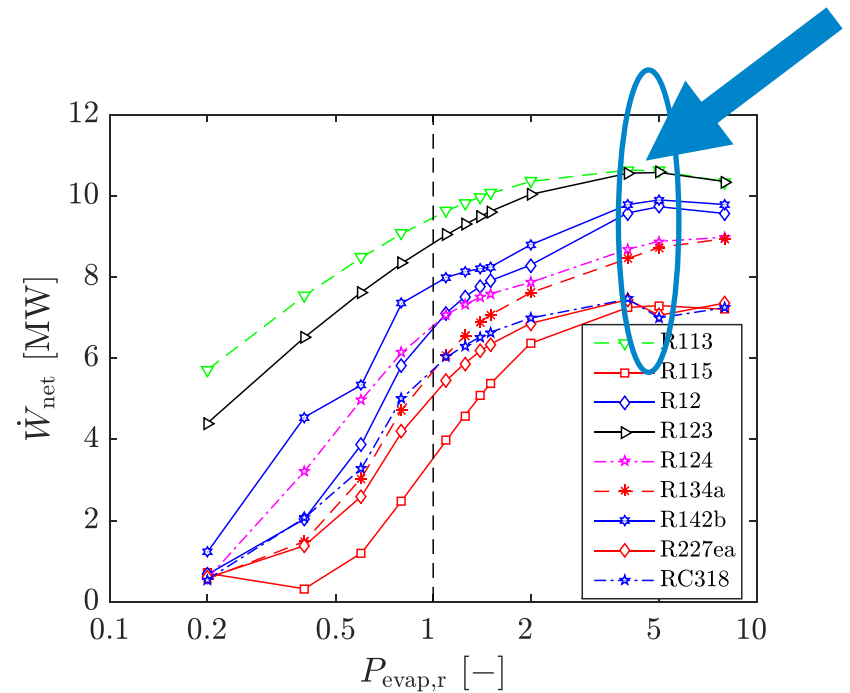
$$P_{\text{evap}} \leq 0.95P_{\text{crit}} \text{ or } P_{\text{evap}} \geq 1.05P_{\text{crit}}$$

$$P_{\text{cond}} \geq 1 \text{ atm} \text{ ***}$$

# Results: Effect of evaporation pressure (1/2)



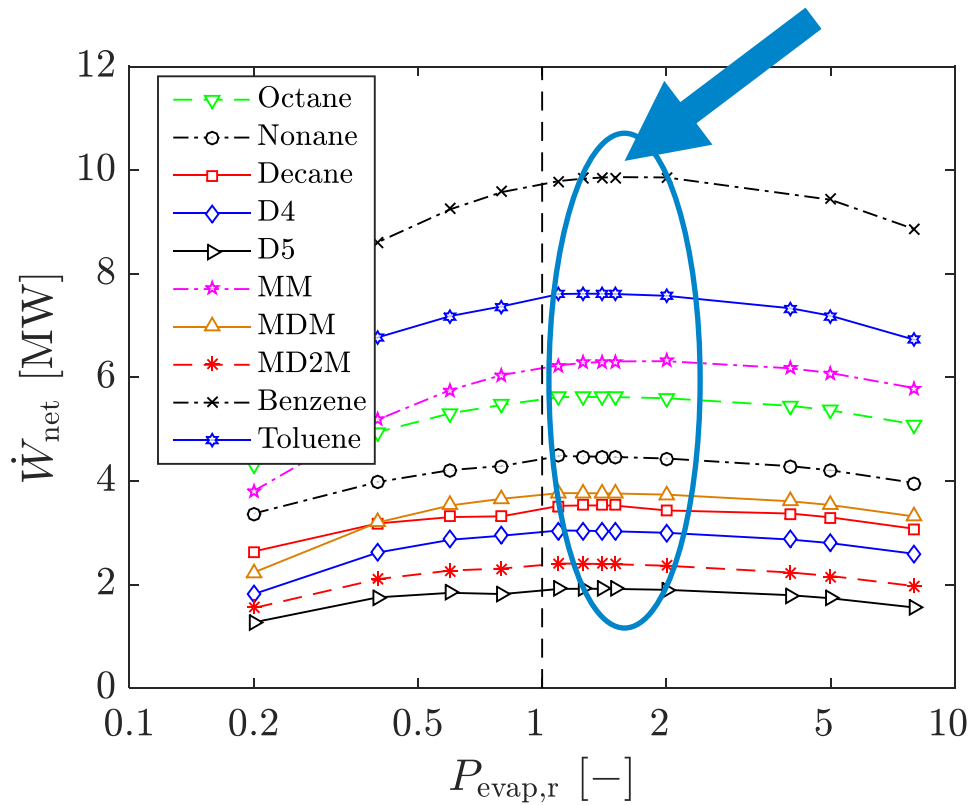
Light hydrocarbons



Refrigerants



## Results: Effect of evaporation pressure (2/2)

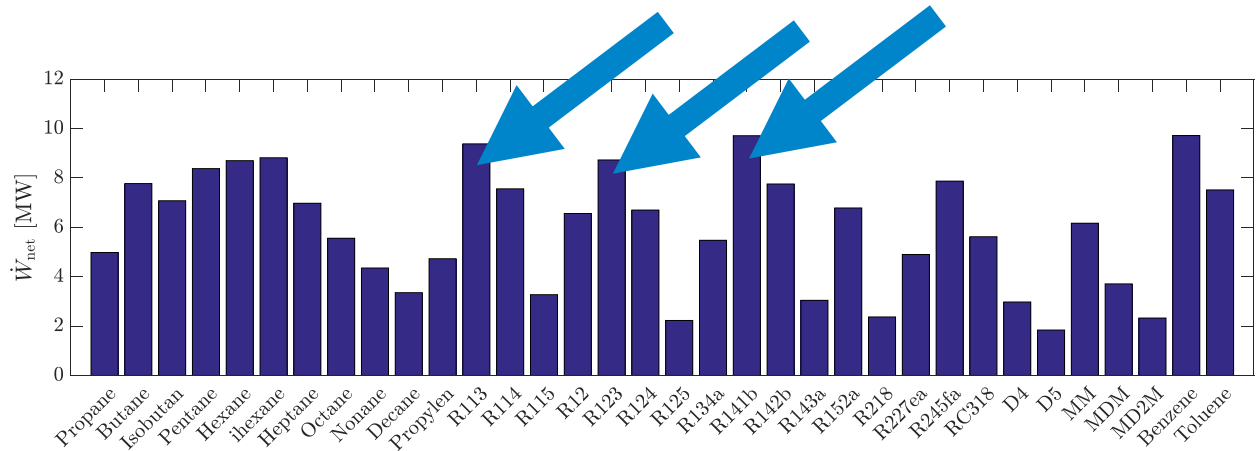


Condensation at  
atmospheric pressure

$$P_{cond} = 1 \text{ atm}$$

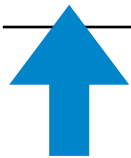
Heavy working fluids

# Results: Recuperators in subcritical cycles



$$P_{\text{evap}} = 0.95P_{\text{crit}}$$

Fluids with $\theta_{\text{recup}} = 0$	Fluids with $0 < \theta_{\text{recup}} < 1$	Fluids with $\theta_{\text{recup}} = 1$
Pentane, R113, R123, R141b	Propane (0.84), propylene (0.70), R115 (0.66), R12 (0.95), R125 (0.37), R134a (0.90), R143a (0.45), R218 (0.64), benzene (0.21), toluene (0.23)	Butane, isobutane, hexane, isohexane, heptane, octane, nonane, R114, R124, R142b, R152a, R227ea, R245fa, RC318, D4, D5, MM, MDM, MD2M



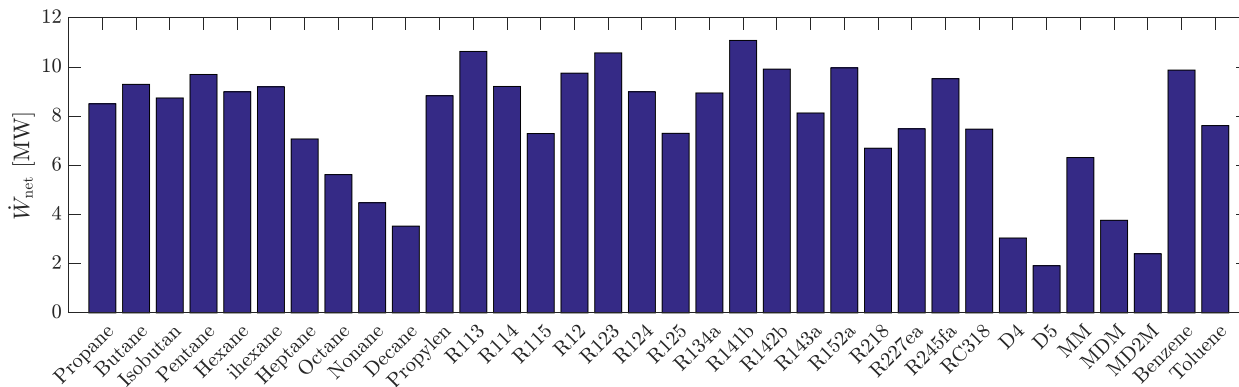
A few slightly dry fluids;  
4 out of 34



- All wet fluids due to superheating required
- Heavy fluids due to condensation limit at 1 atm



# Results: Recuperators in transcritical cycles



~~$P_{\text{evap}} \leq 0.95P_{\text{crit}}$~~

$P_{\text{evap}} \geq 1.05P_{\text{crit}}$

Fluids with  $\theta_{\text{recup}} = 0$

Propane, butane, isobutane, pentane, propylene, R113, R114, R12, R123, R124, R134a, R141b, R142b, R143a, R152a, R245fa

Fluids with  $0 < \theta_{\text{recup}} < 1$

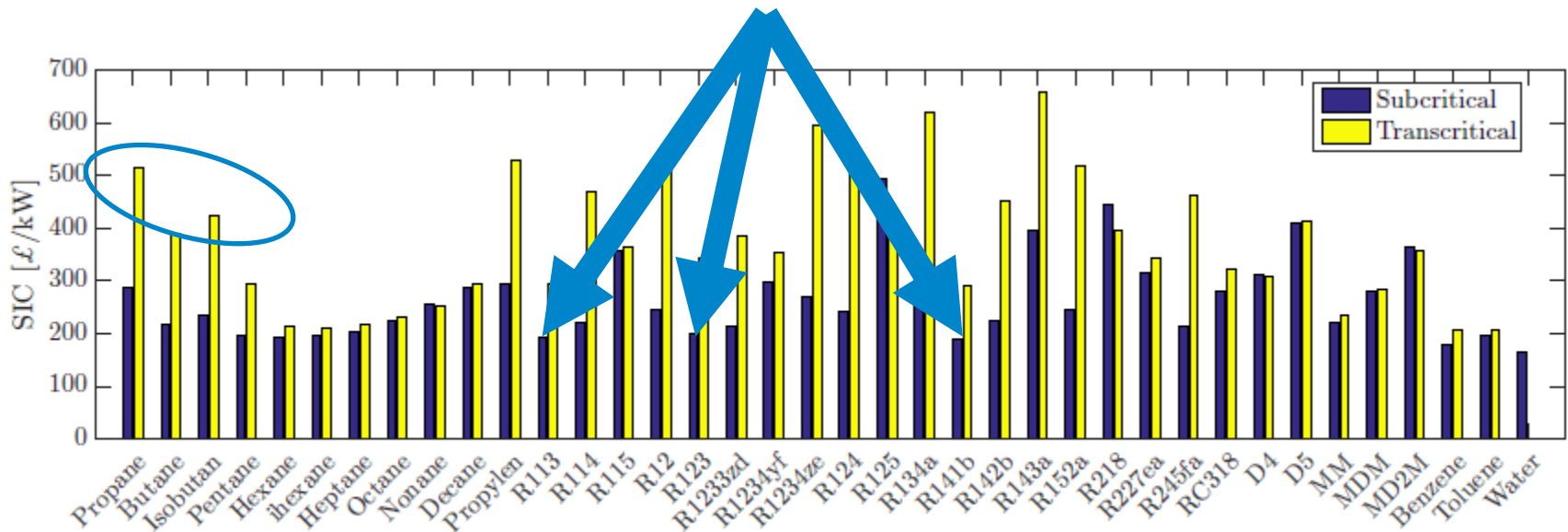
Isohexane (0.80), benzene (0.01), toluene (0.06)

Fluids with  $\theta_{\text{recup}} = 1$

Hexane, heptane, octane, nonane, decane, R115, R125, R218, R227ea, RC318, D4, D5, MM, MDM, MD2M

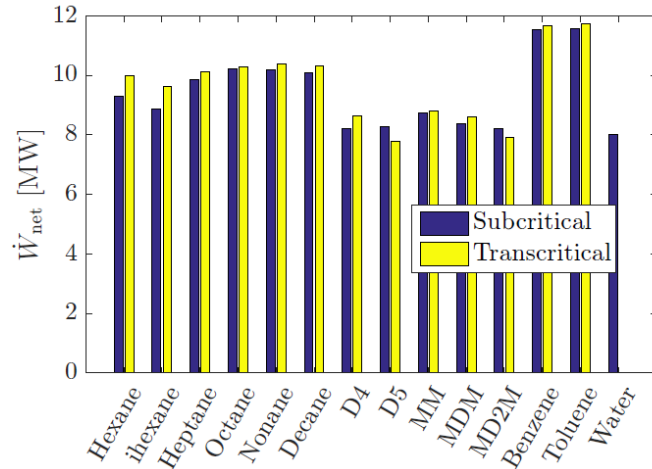
- More fluids (16) no longer require recuperator
- A few refrigerants
- Heavy fluids due to condensation limit at 1 atm

## Results: Costs of sub- and transcritical systems



- Systems with transcritical cycles are generally more expensive
- High pressure systems ( $P_{\text{evap}} \geq 150$  bar) are heavily penalised
- Systems with no recuperators (e.g., R113, R123, R141b) are less expensive

# Results: Relaxing condenser boundary condition



~~$P_{\text{cond}} \geq 1 \text{ atm}^{***}$~~

$$0.002 \text{ bar} \leq P_{\text{cond}} \leq 0.76 \text{ bar}$$

Considerable improvements in power output, up to 200% in some cases

Cycle type	Fluids with $\theta_{\text{recup}} = 0$	Fluids with $\theta_{\text{recup}} \neq 0$
Subcritical	Hexane, isohexane, heptane, octane, nonane, decane, MD2M, benzene, toluene	D4 (0.68), D5 (0.90), MM (1.00), MDM (1.00)
Transcritical	Hexane, isohexane, heptane, octane, nonane, decane, D4, benzene, toluene	D5 (0.70), MM (1.00), MDM (1.00), MD2M (1.00)

- Restriction on condensation pressure lifted
- More fluids no longer require recuperation

## Conclusions

- Case study:
  - Waste-heat recovery from a high-temperature flue-gas stream
  - Over 30 working-fluids; properties from Refprop
  - Objective of maximum power output
- Optimal reduced evaporation pressure (not sub-atmospheric):
  - Transcritical cycles (pentane, R113/123, benzene/toluene)
  - Optimal pressure ratio: 4 – 5 for the refrigerants and light hydrocarbons
  - Optimal pressure ratio: 1 – 2 for heavier hydrocarbons and siloxanes
- Recuperators are required in subcritical systems due to limits on evaporation pressure; transcritical systems rely less on recuperators
- Systems without limits on evaporation/condensation pressure/temperature produce higher power outputs and rely less on the use of recuperators

**Thank you for listening.**

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